

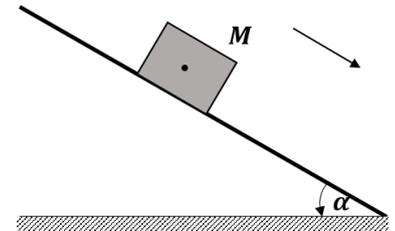
## Tutorial n°5: Dynamics

### Rectilinear and Rotational Motions

#### Exercise 1

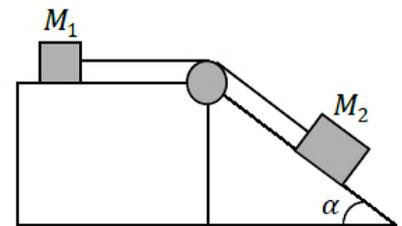
The **figure below** represents a body with a mass  $m$  of  $800\text{ g}$ , moving over a rough plane inclined by  $\alpha$ . The kinetic friction coefficient  $\mu_c$  is  $0.40$ . We take  $g = 10\text{ m s}^{-2}$ .

- 1- Represent the forces acting on the mass  $m$  at the point  $M$ .
- 2- Calculate the angle of inclination so that the body slides with a constant velocity.
- 3- Calculate the normal reaction force  $N$  for  $\alpha = 35^\circ$ .
- 4- Calculate the friction force  $f$  for  $\alpha = 35^\circ$ .
- 5- Calculate the acceleration  $\gamma$  for  $\alpha = 35^\circ$ .



#### Exercise 2

A body  $M_1$  of mass  $m_1 = 5\text{ kg}$ , placed on a horizontal plane, is related to another body  $M_2$  of mass  $m_2$  placed on an inclined plane making an angle  $\alpha = 30^\circ$  with the horizontal, via an inextensible wire of negligible mass and passing through the groove of a pulley of negligible mass (**Figure opposite**). The contact between the two bodies and the two supports (horizontal and inclined) is characterized by static friction coefficient  $\mu_s = 0.3$  and kinetic friction coefficient  $\mu_c = 0.2$ . We will take  $g = 10\text{ m.s}^{-2}$ .

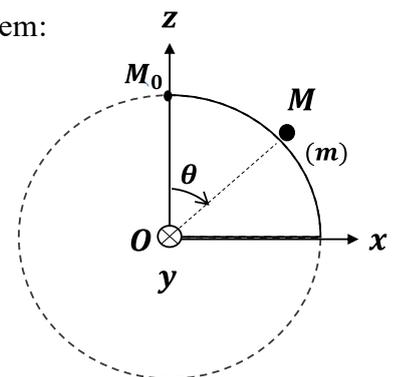


- 1- Determine the maximum value of the mass  $m_{2max}$  necessary to keep the system in equilibrium.
- 2- We take  $m_2 = 7\text{ kg}$ , calculate the **acceleration** of the two bodies and the **tension** of the wire.

#### Exercise 3

We consider a material point of mass ( $m$ ) sliding without frictions on a surface formed by a sphere of radius ( $R$ ) and center  $O$  (**Figure below**). At time  $t = 0\text{ s}$ , the mass is abandoned without initial velocity from the point  $M_0$ , at a point  $M$ . In the intrinsic coordinate system:

- 1- Represent the forces acting on the mass  $m$  at the point  $M$ .
- 2- Establish the equations of movement using the fundamental principle of dynamics (**Newton's 2<sup>nd</sup> law**)
- 3- Demonstrate that the velocity acquired at point  $M$  is given by the expression:  $V(M) = \sqrt{2Rg(1 - \cos\theta)}$ .
- 4- Determine the expression of the contact force  $N$  at the point  $M$ .
- 5- Calculate the angle  $\theta_l$  at which the particle leaves the surface of the sphere and deduce its corresponding velocity  $V_l$ .

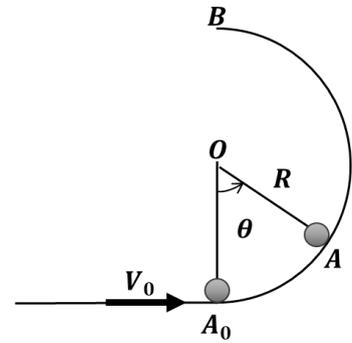


#### Exercise 4

A mobile  $M$  of mass ( $m$ ) is launched with velocity  $V_0$  from point  $A_0$  located at the bottom of a semicircular trajectory of radius ( $R$ ) (**see figure**). All friction forces are neglected.

- 1- Using **2<sup>nd</sup> law of Newton** at point  $M$ , give the equations of the movement in the polar coor-sys.
- 2- Determine the expression of the angular velocity  $\omega_M$ .

- 3- Deduce the expression of linear velocity  $V_M$ .
- 4- Determine the contact force  $N$  exerted by the track on  $A$ .
- 5- Find the condition on  $V_0$  for the particle to arrive at point  $B$ .

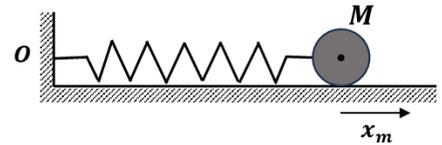


### Exercise 5

A point object  $M$  of mass ( $m$ ), attached to an elastic spring of stiffness constant ( $k$ ) and empty length ( $L_0$ ), moves along a plan. The mass of the spring is assumed to be null and the friction forces are neglected. At equilibrium ( $t=0s$ ), we move this mass away from its equilibrium position by a quantity  $x_m$  and then release it without initial velocity.

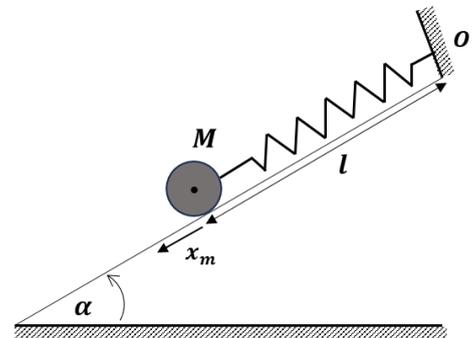
#### a. For horizontal plane

- 1- Represent the forces acting on the mass  $m$ .
- 2- Determine the differential equation of movement by using Newton's 2<sup>nd</sup> law, in the Cartesian coordinate system (Oxyz).
- 3- Deduce the natural oscillation frequency  $\omega_0$ .
- 4- Let  $x(t) = A \sin(\omega_0 t) + B \cos(\omega_0 t)$  be a solution for the differential equation, find the two constants  $A$  and  $B$ .
- 5- Deduce the natural oscillation period  $T_0$  for a system.



#### b. For inclined plane ( $\alpha$ )

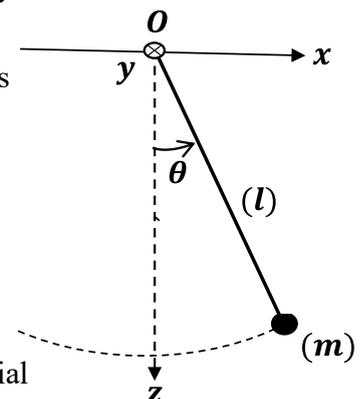
- 1- Represent the forces acting on the mass  $m$  and give their components in the Cartesian coordinate system (Oxyz).
- 2- Establish the equations of movement using Newton's 2<sup>nd</sup> law.
- 3- Find the static equilibrium position  $l$ .
- 4- Deduce the differential equation of movement.



### Exercise 6

We consider a simple pendulum consisting of a material point  $M$  of mass  $m$ , attached to an inextensible wire of length  $l$  and negligible mass. We pull the pendulum at an angle ( $\theta = \theta_0$ ) from its equilibrium position ( $\theta = 0$ ) and we let it go without initial velocity. The friction forces are neglected.

- 1- Represent the forces acting on the point  $M$  and give their components in the polar coordinate system.
- 2- Determine the differential equation of movement in the polar coordinate system using:
  - a. The fundamental principle of dynamics.
  - b. The kinetic moment theorem.
  - c. The mechanical energy theorem.
- 3- Let  $\theta(t) = A \sin(\omega t) + B \cos(\omega t)$  be a solution for the differential equation, find the two constants  $A$  and  $B$  in the case of weak oscillations.



**Exercise-02 :**

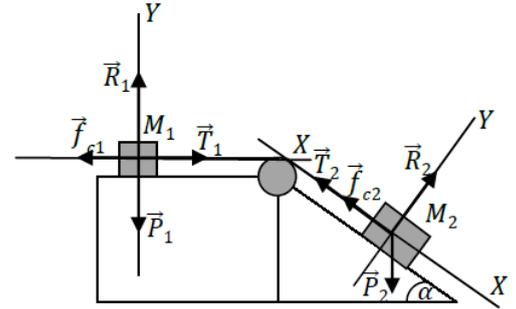
**1/ The maximum value of the mass  $m_{2max}$  necessary to keep the system in equilibrium:**

In equilibrium, we use Newton's 1st law (PI):

$$\Sigma \vec{F}_{ext} = \vec{0} \Rightarrow \begin{cases} (m_1) : \vec{P}_1 + \vec{R}_1 + \vec{T}_1 + \vec{f}_1 = \vec{0} \\ (m_2) : \vec{P}_2 + \vec{R}_2 + \vec{T}_2 + \vec{f}_2 = \vec{0} \end{cases}$$

Projections:  $(m_1) : \begin{cases} (Ox) : T_1 - f_1 = 0 \\ (Oy) : R_1 - P_1 = 0 \end{cases} \Rightarrow \begin{cases} T_1 = f_1 \\ R_1 = P_1 \end{cases}$

$$(m_2) : \begin{cases} (Ox) : P_{2x} - T_2 - f_2 = 0 \\ (Oy) : R_2 - P_{2y} = 0 \end{cases} \Rightarrow \begin{cases} T_2 = P_{2x} - f_2 \\ R_2 = P_{2y} \end{cases}$$



Static friction forces:

$$\begin{cases} f_1 = \mu_s R_1 \\ f_2 = \mu_s R_2 \end{cases} \Rightarrow \begin{cases} f_1 = \mu_s P_1 = \mu_s m_1 g \\ f_2 = \mu_s P_{2y} = \mu_s m_{2max} g \cos \alpha \end{cases}$$

The expression of tensions are :  $\begin{cases} T_1 = f_1 \\ T_2 = P_{2x} - f_2 \end{cases} \Rightarrow \begin{cases} T_1 = \mu_s m_1 g \\ T_2 = m_{2max} g \sin \alpha - \mu_s m_{2max} g \cos \alpha \end{cases}$

An inextensible wire with negligible mass means that:  $T_1 = T_2 = T$

Which allows us to obtain the expression of  $m_{2max}$  :  $m_{2max} = \frac{\mu_s m_1}{\sin \alpha - \mu_s \cos \alpha} = 6.25 \text{ Kg}$

**2/ The acceleration of the two bodies and the tension of the wire if  $m_2 = 7 \text{ kg}$  :**

In motion, we use Newton's 2nd law (FPD):

$$\Sigma \vec{F}_{ext} = m \vec{\gamma} \Rightarrow \begin{cases} (m_1) : \vec{P}_1 + \vec{R}_1 + \vec{T}_1 + \vec{f}_1 = m_1 \vec{\gamma} \\ (m_2) : \vec{P}_2 + \vec{R}_2 + \vec{T}_2 + \vec{f}_2 = m_2 \vec{\gamma} \end{cases}$$

Projections:  $(m_1) : \begin{cases} (Ox) : T_1 - f_1 = m_1 \gamma \dots \dots \dots (1) \\ (Oy) : R_1 - P_1 = 0 \end{cases}$

$$(m_2) : \begin{cases} (Ox) : P_{2x} - T_2 - f_2 = m_2 \gamma \dots \dots (2) \\ (Oy) : R_2 - P_{2y} = 0 \end{cases}$$

Kinetic friction forces :

$$\begin{cases} f_1 = \mu_c R_1 \\ f_2 = \mu_c R_2 \end{cases} \Rightarrow \begin{cases} f_1 = \mu_c P_1 = \mu_c m_1 g \\ f_2 = \mu_c P_{2y} = \mu_c m_2 g \cos \alpha \end{cases}$$

The acceleration  $\gamma$  will be determined from eq (1) and (2):  $\begin{cases} eq(3) : m_1 \gamma = T_1 - f_1 \\ eq(4) : m_2 \gamma = P_{2x} - T_2 - f_2 \end{cases}$

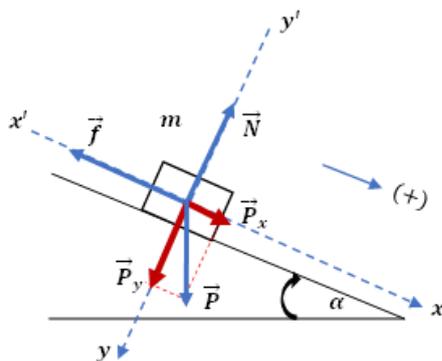
$$\Rightarrow \begin{cases} m_1 \gamma = T - \mu_c m_1 g \\ m_2 \gamma = m_2 g \sin \alpha - T - \mu_c m_2 g \cos \alpha \end{cases} \Rightarrow m_1 \gamma + m_2 \gamma = m_2 g \sin \alpha - \mu_c m_2 g \cos \alpha - \mu_c m_1 g$$

$$\gamma = \frac{m_2 (g \sin \alpha - \mu_c g \cos \alpha) - \mu_c m_1 g}{m_1 + m_2} = 1.07 \text{ m.s}^{-2}$$

According to eq (1), the tension of the wire is:  $T = m_1 \gamma + \mu_c m_1 g = 15.35 \text{ N}$

**Exercise-01**

**1/ Representation of forces**



**3/ Angle of inclination alpha**

The body moves at constant velocity, this means that the sum of the forces must be zero (principle of inertia, Newton's 1st law):

$$V = C^{te} \Rightarrow \sum \vec{F}_{ext} = \vec{0}$$

$$\vec{P} + \vec{f} + \vec{N} = \vec{0}$$

By projection on the two axes, it comes:

$$\begin{cases} P_x - f = 0 \\ p_y - N = 0 \end{cases} \Rightarrow \begin{cases} P_x = f \\ p_y = N \end{cases} \Rightarrow \begin{cases} p \sin \alpha = \mu N \\ p \cos \alpha = N \end{cases}$$

$$\Rightarrow \frac{p \sin \alpha}{p \cos \alpha} = \frac{\mu N}{N} \Rightarrow \tan \alpha = \mu \Rightarrow \alpha = 21.8^\circ$$

**4/ The normal reaction force N for alpha = 35°**

$$p \cos \alpha = N \Rightarrow N = mg \cos \alpha = 8 \cos 35$$

$$N = 6.55 N$$

**3/ The friction force f for alpha = 35°**

$$f = \mu N \Rightarrow f = 2.62 N$$

**5/ Acceleration for alpha = 35°**

Angle of inclination 35° > 21.8° (case of constant Velocity), this means that the Velocity is variable (fundamental principle of dynamics, Newton's 2<sup>nd</sup> law):

$$PFD \Rightarrow \sum \vec{F}_{ext} = m\vec{\gamma}$$

By projection on the two axes, it comes:

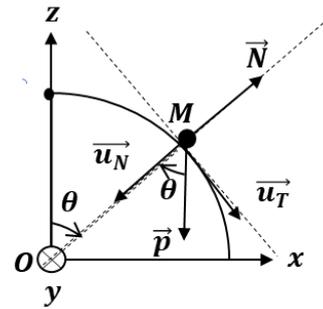
$$\begin{cases} P_x - f = m\gamma \\ p_y - N = 0 \end{cases} \Rightarrow p \sin \alpha - f = m\gamma$$

$$\Rightarrow \gamma = \frac{mg \sin \alpha - f}{m} \Rightarrow \gamma = 2.46 \text{ m/s}^2$$

**Exercise-03**

**1/ The forces acting on the mass m at the point M**

The forces acting on point M are:  $\vec{P}$  (the weight force) and  $\vec{N}$  (the contact force).



**2/ Equations of movement**

Using the fundamental principle of dynamics (Newton's 2nd law (FPD)), in the intrinsic coordinate system:

$$\sum \vec{F}_{ext} = m\vec{\gamma}$$

$$\Rightarrow \vec{P} + \vec{N} = m\vec{\gamma}$$

By projecting the forces onto the tangential (T) and the normal (N) axes, we determine their component:

$$\vec{P} \begin{pmatrix} mg \sin \theta \\ mg \cos \theta \end{pmatrix}; \vec{N} \begin{pmatrix} 0 \\ -N \end{pmatrix}$$

$$\text{So: } \vec{P} \begin{pmatrix} mg \sin \theta \\ mg \cos \theta \end{pmatrix} + \vec{N} \begin{pmatrix} 0 \\ -N \end{pmatrix} = m\vec{\gamma} \begin{pmatrix} \gamma_T \\ \gamma_N \end{pmatrix}$$

$$\text{knowing that: } \begin{cases} \gamma_T = \frac{dV}{dt} \\ \gamma_N = \frac{V^2}{R} \end{cases}$$

The equations of movement will be as follows:

$$\Rightarrow \begin{cases} mg \sin \theta + 0 = m \frac{dV}{dt} \dots \dots (1) \\ mg \cos \theta - N = m \frac{V^2}{R} \dots \dots (2) \end{cases}$$

**3/ Show that  $V(M) = \sqrt{2Rg(1 - \cos \theta)}$ .**

According to eq (1), we have:

$$mg \sin \theta = m \frac{dV}{dt}$$

$$\Rightarrow g \sin \theta = \frac{dV}{d\theta} \frac{d\theta}{dt} = \frac{dV}{d\theta} \frac{V}{R}$$

$$\text{and } \begin{cases} \omega = \frac{V}{R} : \text{angular velocity, because} \\ V = \frac{ds}{dt} = R \frac{d\theta}{dt} = R\omega \\ \omega = \frac{d\theta}{dt} \end{cases}$$

$$\Rightarrow \frac{d\theta}{dt} = \frac{V}{R}$$

$$\Rightarrow V \cdot dV = g R \sin \theta \cdot d\theta$$

$$\Rightarrow \int_0^V V \cdot dV = g R \int_0^\theta \sin \theta \cdot d\theta$$

$$\Rightarrow V(M) = \sqrt{2Rg(1 - \cos \theta)}$$

**4/ Expression of the contact force N**

According to eq (2), we have:

$$mg \cos \theta - N = m \frac{V^2}{R}$$

$$\Rightarrow N = mg \cos \theta - m \frac{V^2}{R}$$

$$\Rightarrow N = mg \cos \theta - 2mg (1 - \cos \theta)$$

$$N = mg (3 \cos \theta - 2)$$

**5/ Angle  $\theta_L$  and velocity  $V_L$  at which the particle leaves the surface of the sphere**

When the particle leaves the surface of the sphere: the contact force N is canceled, we write:

$$\Rightarrow N = mg (3 \cos \theta - 2) = 0$$

$$\Rightarrow \cos \theta = \frac{2}{3}$$

$$\Rightarrow \theta_L = \arccos \left( \frac{2}{3} \right) = 48^\circ$$

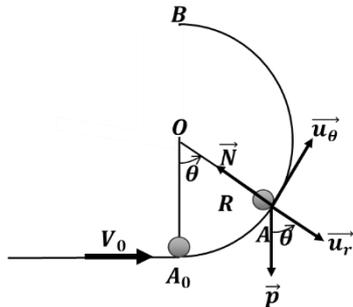
The corresponding velocity  $V_L$  is:

$$V = \sqrt{2Rg (1 - \cos(48^\circ))} = \sqrt{2Rg (1 - 2/3)}$$

$$\Rightarrow V_L = \sqrt{\frac{2}{3} Rg}$$

**Exercise-04**

**1/ Equations of movement**



According to the Newton's 2nd law (PDF):

$$\sum \vec{F}_{ext} = m\vec{\gamma} \Rightarrow \vec{P} + \vec{N} = m\vec{\gamma}$$

By projection on the radial and angular axes:

$$\vec{P} \begin{cases} mg \cos \theta \\ -mg \sin \theta \end{cases} + \vec{N} \begin{cases} -N \\ 0 \end{cases} = m\vec{\gamma} \begin{cases} \gamma_r \\ \gamma_\theta \end{cases}$$

$$\begin{cases} \gamma_r = \frac{d^2r}{dt^2} - r \left( \frac{d\theta}{dt} \right)^2 \\ \gamma_\theta = 2 \frac{dr}{dt} \frac{d\theta}{dt} + r \frac{d^2\theta}{dt^2} \end{cases} \xrightarrow{r=R=cst} \begin{cases} \gamma_r = -R \left( \frac{d\theta}{dt} \right)^2 \\ \gamma_\theta = R \frac{d^2\theta}{dt^2} \end{cases}$$

The equations of movement will be as follows:

$$\Rightarrow \begin{cases} mg \cos \theta - N = -mR \left( \frac{d\theta}{dt} \right)^2 \dots \dots (1) \\ -mg \sin \theta = mR \frac{d^2\theta}{dt^2} \dots \dots \dots (2) \end{cases}$$

**2/ Angular velocity  $\omega_M$**

According to eq (2), we have:

$$-mg \sin \theta = mR \frac{d}{dt} \left( \frac{d\theta}{dt} \right); \quad \omega = \frac{d\theta}{dt}$$

$$\Rightarrow -mg \sin \theta = mR \frac{d\omega}{dt} \frac{d\theta}{d\theta}$$

$$\Rightarrow \int_{\omega_0}^{\omega} \omega d\omega = -\frac{g}{R} \int_0^\theta \sin \theta d\theta$$

$$\Rightarrow \omega_M = \sqrt{\omega_0^2 - \frac{2g}{R} (1 - \cos \theta)}$$

**3/ Linear velocity  $V_M$**

We have:  $\omega = \frac{V}{R} \Rightarrow V_M = \sqrt{V_0^2 - 2gR(1 - \cos \theta)}$

**4/ Contact force N**

According to eq (1), we have:

$$mg \cos \theta - N = -mR \left( \frac{d\theta}{dt} \right)^2$$

$$N = mg (2 - \cos \theta - \omega_0^2 R)$$

**5/ Velocity  $V_0$**

For mobile M to arrive at point B, it must:

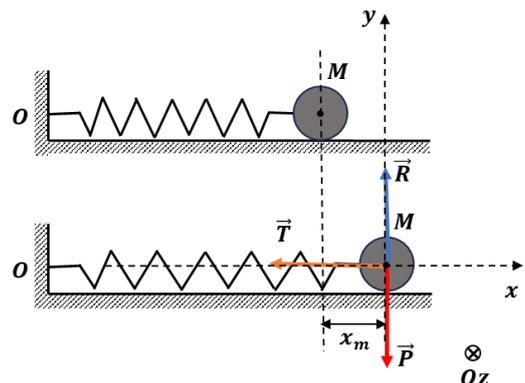
$$V_M > 0 \text{ and } \theta = \pi \Rightarrow V_0^2 - 2gR(1 - \cos \theta) > 0$$

$$V_0 > \sqrt{4gR}$$

**Exercise-05**

**a. For horizontal plane**

**1) Representation of forces**



## 2) Differential equation of movement

using 2<sup>nd</sup> law of Newton:

$$\sum \vec{F}_{ext} = m \vec{\gamma}$$

$$\vec{P} + \vec{R} + \vec{T} = m \vec{\gamma}$$

By projection:

$$\{ Ox: -T = m\gamma \dots \dots (1)$$

$$\{ Oy: R - P = 0 \dots \dots (2)$$

From eq (1):  $-T = m\gamma$

$$\Rightarrow -kx = m \frac{d^2x}{dt^2}$$

$$\frac{d^2x}{dt^2} + \frac{k}{m}x = 0$$

$$\text{or: } \ddot{x} + \frac{k}{m}x = 0$$

it presents the differential equation of 2<sup>nd</sup> order, which takes the form:  $\ddot{x} + \omega_0 x = 0$

### 3) Natural oscillation frequency $\omega_0$

From the form:  $\ddot{x} + \omega_0 x = 0$

$$\omega_0 = \sqrt{\frac{k}{m}}$$

### 4) Constants A and B

We have:  $x(t) = A \sin(\omega_0 t) + B \cos(\omega_0 t)$

And:  $\dot{x}(t) = A\omega_0 \cos(\omega_0 t) - B\omega_0 \sin(\omega_0 t)$

From the initial conditions:  $\begin{cases} x(t=0) = x_m \\ \dot{x}(t=0) = 0 \end{cases}$

Therefore:  $\begin{cases} x(0) = B = x_m \\ \dot{x}(0) = A\omega_0 \Rightarrow \begin{cases} B = x_m \\ A = 0 \end{cases} \end{cases}$

Solution:  $x(t) = x_m \cos\left(\sqrt{\frac{k}{m}}t\right)$

### 5) Natural oscillation period $T_0$

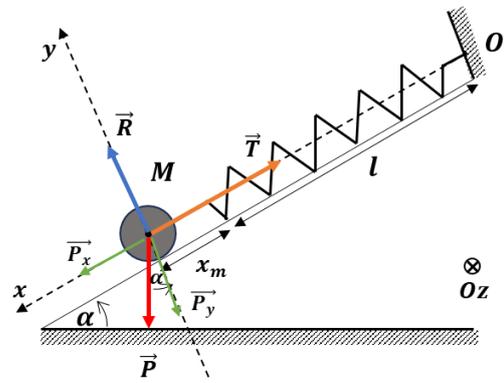
$$T_0 = \frac{2\pi}{\omega_0}$$

$$T_0 = 2\pi \sqrt{\frac{m}{k}}$$

#### b. For inclined plane ( $\alpha$ )

##### 1) Representation of forces

Components:  $\vec{P} \begin{pmatrix} mg \sin \alpha \\ mg \cos \alpha \\ 0 \end{pmatrix}; \vec{R} \begin{pmatrix} 0 \\ R \\ 0 \end{pmatrix}; \vec{T} \begin{pmatrix} -T \\ 0 \\ 0 \end{pmatrix}$



## 2) Equations of movement

using 2<sup>nd</sup> law of Newton:

$$\sum \vec{F}_{ext} = m \vec{\gamma}$$

$$\vec{P} + \vec{R} + \vec{T} = m \vec{\gamma}$$

By projection:

$$\{ Ox: mg \sin \alpha - T = m\gamma \dots \dots (1)$$

$$\{ Oy: -mg \cos \alpha + R = 0 \dots \dots (2)$$

### 3) Static equilibrium position l

At equilibrium:  $V = 0 \Rightarrow \gamma = 0$

From eq (1):  $mg \sin \alpha - T = 0$

$$\Rightarrow mg \sin \alpha = k(\Delta l)$$

$$\Rightarrow mg \sin \alpha = k(l - l_0)$$

$$\Rightarrow l = l_0 + \frac{mg}{k} \sin \alpha$$

### 3) Differential equation of movement

From eq (1):  $mg \sin \alpha - T = m\gamma$

$$\Rightarrow mg \sin \alpha - k(\Delta l + x) = m \frac{d^2x}{dt^2}$$

$$\Rightarrow kx = m \frac{d^2x}{dt^2}$$

Because:  $mg \sin \alpha = k(\Delta l)$

$$\Rightarrow \frac{d^2x}{dt^2} + \frac{k}{m}x = 0$$

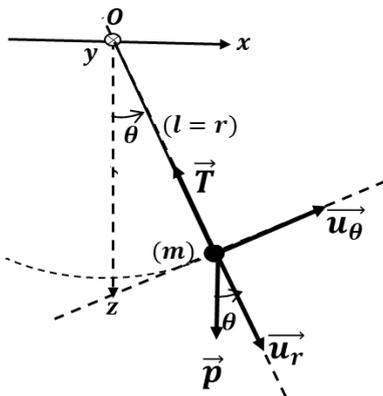
$$\Rightarrow \ddot{x} + \frac{k}{m}x = 0$$

We obtain the same differential equation of 2<sup>nd</sup> order, which takes the form:  $\ddot{x} + \omega_0 x = 0$  and having the same

solution:  $x(t) = x_m \cos\left(\sqrt{\frac{k}{m}}t\right)$

**Exercise-06**

**1/ The forces acting on the point M and their components in the polar R.**



The forces acting on point M are:  $\vec{P}$  (the weight force) and  $\vec{T}$  (the tension force). By projecting the forces onto the radial (r) and the angular ( $\theta$ ) axes we determine their component:

$$\vec{P} \begin{pmatrix} mg \cos \theta \\ -mg \sin \theta \end{pmatrix}; \vec{T} \begin{pmatrix} -T \\ 0 \end{pmatrix}$$

**2/ Differential Equation of movement**

**a. By applying the fundamental principle of dynamics (FPD)**

According to Newton's 2nd law (PDF):

$$\sum \vec{F}_{ext} = m\vec{\gamma} \\ \Rightarrow \vec{P} + \vec{T} = m\vec{\gamma}$$

By projection on the radial and angular axes:

$$\vec{P} \begin{pmatrix} mg \cos \theta \\ -mg \sin \theta \end{pmatrix} + \vec{T} \begin{pmatrix} -T \\ 0 \end{pmatrix} = m\vec{\gamma} \begin{pmatrix} \gamma_r \\ \gamma_\theta \end{pmatrix}$$

$$\begin{cases} \gamma_r = \frac{d^2r}{dt^2} - r \left(\frac{d\theta}{dt}\right)^2 \\ \gamma_\theta = 2 \frac{dr}{dt} \frac{d\theta}{dt} + r \frac{d^2\theta}{dt^2} \end{cases} \xrightarrow{r=l=cst} \begin{cases} \gamma_r = -l \left(\frac{d\theta}{dt}\right)^2 \\ \gamma_\theta = l \frac{d^2\theta}{dt^2} \end{cases}$$

The equations of movement will be as follows:

$$\Rightarrow \begin{cases} mg \cos \theta - T = -ml \left(\frac{d\theta}{dt}\right)^2 \dots \dots (1) \\ -mg \sin \theta = ml \frac{d^2\theta}{dt^2} \dots \dots \dots (2) \end{cases}$$

According to eq (2), we write:

$$-mg \sin \theta = ml \frac{d^2\theta}{dt^2} \Rightarrow -g \sin \theta = l \frac{d^2\theta}{dt^2}$$

The differential equation of movement is therefore written:

$$\frac{d^2\theta}{dt^2} + \frac{g}{l} \sin \theta = 0$$

**b. By applying the kinetic moment theorem**

Kinetic moment theorem: the derivative of kinetic moment about a point O is equal to the sum of the moments of the external forces applied to material point M about a point O.

$$\frac{d\vec{L}_O}{dt} = \sum \vec{M}_O(\vec{F})$$

$$\frac{d\vec{L}_O}{dt} = \vec{M}_O(\vec{P}) + \vec{M}_O(\vec{T}) \dots \dots \dots \text{eq} (*)$$

\*Let's calculate the kinetic moment  $\vec{L}_O$ :

$$\vec{L}_O = \vec{OM} \wedge \vec{p}$$

$$\text{where: } \begin{cases} \vec{OM} = l \vec{u}_r \\ \vec{p} = m \vec{V} = m \left( \frac{dr}{dt} \vec{u}_r + r \frac{d\theta}{dt} \vec{u}_\theta \right) \end{cases}$$

knowing that:  $r = l = cst$

$$\begin{cases} \vec{OM} = l \vec{u}_r \\ \vec{p} = ml \frac{d\theta}{dt} \vec{u}_\theta \end{cases} \Rightarrow \vec{L}_O = \vec{OM} \wedge \vec{p} = ml^2 \frac{d\theta}{dt} \vec{k}$$

$$\vec{L}_O = ml^2 \frac{d\theta}{dt} \vec{k}$$

$$\frac{d\vec{L}_O}{dt} = ml^2 \frac{d^2\theta}{dt^2} \vec{k} \dots \dots (3)$$

Let's calculate the moment of forces:  $\vec{M}_O(\vec{P}), \vec{M}_O(\vec{T})$

$$\vec{M}_O(\vec{P}) = \vec{OM} \wedge \vec{P}$$

$$\text{where: } \begin{cases} \vec{OM} = l \vec{u}_r \\ \vec{P} = mg \cos \theta \vec{u}_r - mg \sin \theta \vec{u}_\theta \end{cases} \\ \Rightarrow \vec{M}_O(\vec{P}) = -mgl \sin \theta \vec{k} \dots \dots (4)$$

$$\vec{M}_O(\vec{T}) = \vec{OM} \wedge \vec{T} \Rightarrow \begin{cases} \vec{OM} = l \vec{u}_r \\ \vec{T} = -T \vec{u}_r + 0 \vec{u}_\theta \end{cases} \\ \Rightarrow \vec{M}_O(\vec{T}) = \vec{0} \dots \dots \dots (5)$$

We replace eqs (3, 4, 5) in (\*):

$$ml^2 \frac{d^2\theta}{dt^2} \vec{k} = -mgl \sin \theta \vec{k}$$

The differential equation of movement is therefore written:

$$\frac{d^2\theta}{dt^2} + \frac{g}{l} \sin \theta = 0$$

**c. By applying the mechanical energy theorem**

$$E_m = E_c + E_p$$

$$\Rightarrow E_m = \frac{1}{2} m V^2 + mgh$$

$$\text{knowing that: } \begin{cases} V = r \frac{d\theta}{dt} = l \frac{d\theta}{dt} \\ h = l(1 - \cos \theta) \end{cases}$$

$$\Rightarrow E_m = \frac{1}{2} m l^2 \left( \frac{d\theta}{dt} \right)^2 + mgl - mgl \cos \theta$$

As long as there are no friction forces, the mechanical energy is therefore conserved, i.e.:

$$\begin{aligned} \frac{dE_m}{dt} &= 0 \\ \Rightarrow \frac{dE_m}{dt} &= m l^2 \frac{d\theta}{dt} \frac{d^2\theta}{dt^2} + mgl \sin \theta \frac{d\theta}{dt} = 0 \end{aligned}$$

The differential equation of movement is therefore written:

$$\Rightarrow \frac{d^2\theta}{dt^2} + \frac{g}{l} \sin \theta = 0$$

### 3/ Equation of movement $\theta(t)$

For very small amplitude oscillations  $\sin \theta \approx \theta$ , the differential equation is written in the form:

$$\Rightarrow \frac{d^2\theta}{dt^2} + \frac{g}{l} \theta = 0$$

This differential equation is of the 2nd degree, with the form:  $y'' + \omega^2 y = 0$ , it has a solution of the form:

$$\theta(t) = A \sin(\omega t) + B \cos(\omega t), \quad \omega = \sqrt{\frac{g}{l}}$$

To determine A and B, we use the initial conditions:

$$\text{at } t = 0 \text{ s : } \begin{cases} \theta = \theta_0 \\ V_0 = 0 \end{cases} \Rightarrow \frac{d\theta}{dt} = \frac{V_0}{r} = 0$$

$$\text{we have : } \begin{cases} \theta(t) = A \sin(\omega t) + B \cos(\omega t) \\ \dot{\theta} = A\omega (\cos(\omega t) - B\sin(\omega t)) \end{cases}$$

So: at  $t = 0$  s :

$$\begin{cases} \theta(0) = A \sin(0) + B \cos(0) = \theta_0 \\ \dot{\theta} = A\omega (\cos(0) - B\sin(0)) = 0 \end{cases}$$

So:  $\begin{cases} B = \theta_0 \\ A = 0 \end{cases}$  we replace them in the solution  $\theta(t)$ :

$$\theta(t) = \theta_0 \cos\left(\sqrt{\frac{g}{l}} t\right)$$